

(P11) A crank OA, 100 mm long, rotates clockwise at 100 rev/min, as shown in the figure. Rod AC, 500 mm long, slides in a swivelling pin at B. The end C slides on a swinging link DE. When the angle BOA is 120° , find the angular velocity and angular accⁿ of DE.

Solution: -

Velocity diagram -

1) $V_{ao} = \omega(AO) \omega_{oa} = (0.1) \times \frac{2\pi(100)}{60} = 1.05 \text{ m/s}$

Draw V_{ao} .

2) Assume B' to be coincident with B, and to be on link AC.

$V_{b'a}$ will be \perp to AB. Through 'a', draw a line \perp to AB. $V_{b'b}$ will be \parallel to AB'. Through 'b', draw a line \parallel to AB'. The intersection of these two lines locates point 'b'.

3) Extend 'ab' to 'c' such that $\frac{b'c}{ab'} = \frac{B'C}{AB'}$

$$\therefore b'c = \left(\frac{137.5}{362.5} \right) \cdot (3.65) \text{ cm}$$

$$\therefore b'c = 1.38 \text{ cm}$$

Thus, 'c' can be located.

4) Assume c' to be coincident with c, but on the link DE. $V_{c'd}$ will be \parallel to DC'. Through 'c', draw a line \parallel to DC'. $V_{c'd}$ will be \perp to DC'. Through 'd', draw a line \perp to DC'. The intersection of these two lines locates point 'c'.

Thus, the velocity diagram can be completed.

From the velocity diagram,

$$\begin{aligned} \text{Ang. velocity of link DE} \\ = \omega_{de} &= \frac{V_{c'd}}{\omega(C'D)} \quad (\because c' \text{ is on link DE}) \\ &= \frac{0.576}{0.12} \end{aligned}$$

$$\therefore \omega_{de} = 4.8 \text{ rad/s} \quad \curvearrowright = \omega_{cd}$$

Acceleration diagram

Sr. No.	Vector	Magnitude	Direction	Sense
1	a_{ao}^c	$= \frac{V_{ao}^2}{r(AO)} = \frac{(1.05)^2}{(0.1)} = 11.03 \text{ m/s}^2$	to OA	Towards O in Config. diag.
2	a_{ao}^t	$= 0 \quad \therefore \alpha_{ao} = 0$	—	—
3	a_{ba}^c	$= \frac{V_{ba}^2}{r(B'A)} = \frac{(0.73)^2}{(0.3625)} = 1.47 \text{ m/s}^2$	to B'A	Towards A in config. diagram
4	a_{ba}^t	$= r(AB)\alpha_{ab} = 4.75 \text{ m/s}^2$ $\therefore \alpha_{ab} = \frac{4.75}{0.3625} = 13.10 \text{ rad/s}^2$	\perp to a_{ba}^c	—
5	a_{bb}^s (sliding)	$= 6.3 \text{ m/s}^2$	to B'A	—
6	a_{bb}^{cor} (Coriolis)	$= 2 V_{bb} \omega_{ba}$ $= 2(0.74)(2.01)$ $= 2.98 \text{ m/s}^2$	Now $\omega_{ba} = \frac{V_{ba}}{r(B'A)} = \frac{0.73}{(0.3625)} = 2.01 \frac{\text{rad}}{\text{s}}$ As shown on Config. diag.	—
7	a_{cd}^c	$= \frac{V_{cd}^2}{r(cd)} = \frac{(0.576)^2}{(0.12)} = 2.77 \text{ m/s}^2$	to c'D	Towards D in Config. diag.
8	a_{cd}^t	$= r(c'D)\alpha_{cd} = 10 \text{ m/s}^2$ $\therefore \alpha_{cd} = \frac{10}{0.12} = 83.33 \text{ rad/s}^2$	\perp to a_{cd}^c	—
9	a_{cc}^s (sliding)	$= 0.65 \text{ m/s}^2$	to c'D	—
10	a_{cc}^{cor} (Coriolis)	$= 2 V_{cc} \omega_{cd} = 2(0.54)(4.8)$ $= 5.18 \text{ m/s}^2$	As shown on config. diag.	—

* The vectors are drawn serially as above.

The intersection \rightarrow
 Sr. No. 1, 2, 3 are drawn. Sr. No. 4 is drawn in direction.
 Sr. No. 6 is drawn. Sr. No. 5 is drawn in direction.

The intersection of Sr. No. 4 & 5 gives point 'b'.

'ab' is now extended to give point 'c' such that

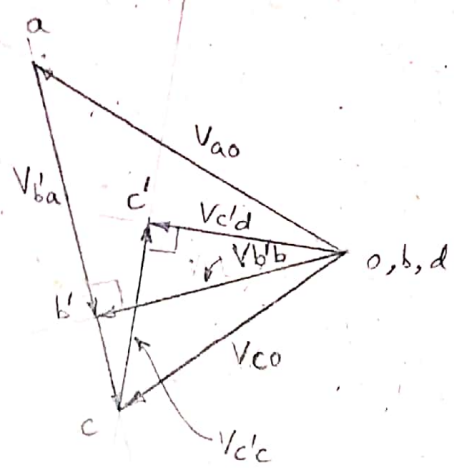
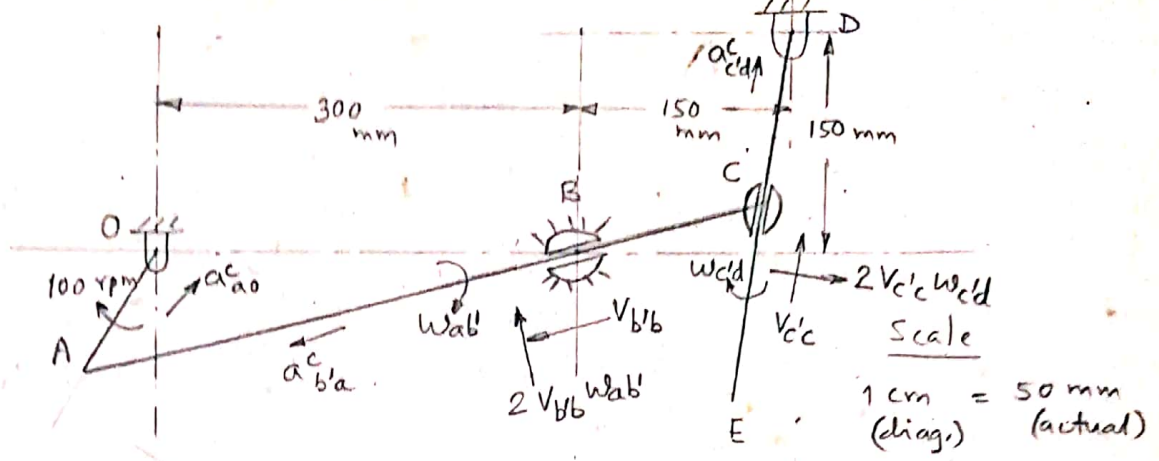
$$\frac{bc}{ab} = \frac{Bc}{AB} \quad \therefore bc = \left(\frac{5}{cm}\right) \left(\frac{137.5}{362.5}\right) \quad \therefore bc = 1.9 \text{ cm.}$$

Thus, point 'c' is located.

Then Sr. No. 7 is drawn. Sr. No. 8 is drawn in direction. Then Sr. No. 10 is drawn. Sr. No. 9 is drawn in direction. The intersection of Sr. No. 8 & 9 gives point 'c'.

From Accⁿ diagram, $\alpha_{de} = \alpha_{cd} = 83.33 \text{ rad/s}^2$

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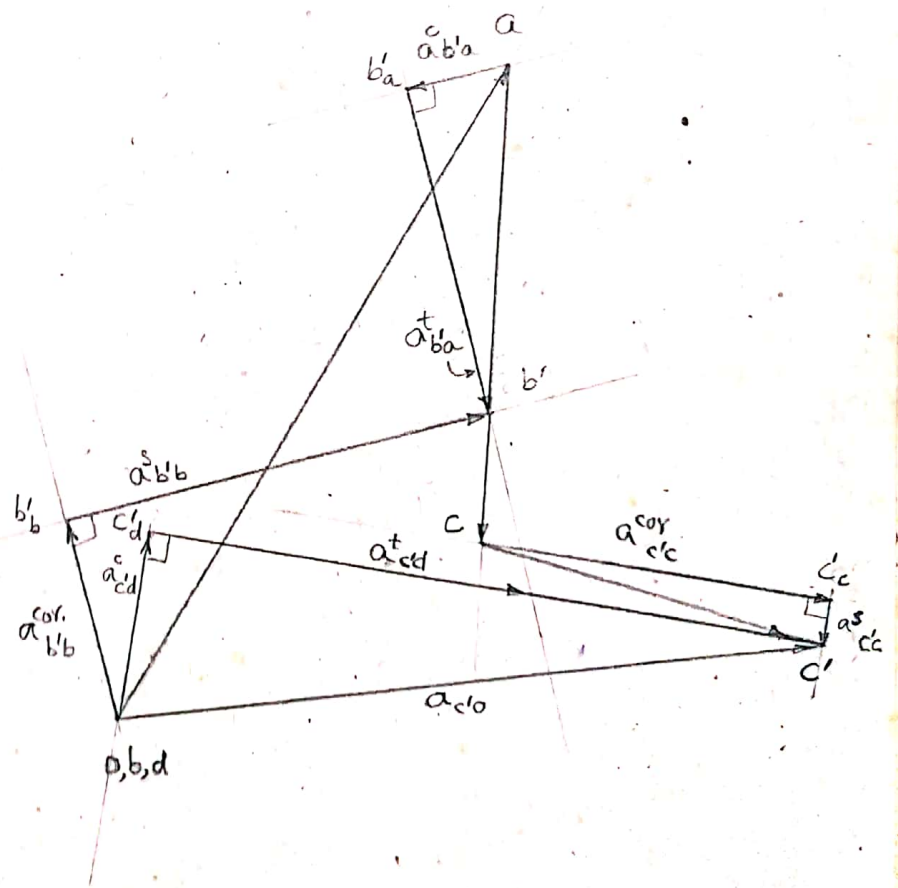


Vel. diagram

1 cm = 0.2 m/s (diag) 1 cm = 50 mm (actual)

Acc'n diagram

1 cm = 1 m/s² (diag) 1 cm = 50 mm (actual)



P12 The fig. shows a quick return mechanism, in which the driving crank OA rotates at 120 rpm in a clockwise direction. For the position shown, determine the magnitude and direction of

- The accⁿ of block D.
- The angular accⁿ of the slotted bar QB.

Solution -

Velocity diagram

$$1) V_{ao} = \omega(OA) \left(\frac{2\pi(120)}{60} \right) = 2.51 \text{ m/s.}$$

$$\text{Draw } V_{ao}. \quad \begin{matrix} 1 \text{ cm} & = & 1 \text{ m/s} \\ \text{(diag.)} & & \text{(actual)} \end{matrix}$$

- Assume A_1 to be coincident with A, but on link BC.

~~V_{a_1q} will be \perp to AA_1 . Through~~

V_{a_1a} will be \parallel to A_1Q . Through 'a', draw a line \parallel to A_1Q . V_{a_1q} will be \perp to A_1Q . Through 'q', draw a line \perp to A_1Q . The intersection locates point 'a'.

- Extend a_1q to point 'c', such that

$$\frac{qc}{a_1q} = \frac{QC}{A_1Q} \quad \therefore qc = \frac{(2.45)}{\text{cm}} \left(\frac{150}{262.5} \right) = 1.4 \text{ cm.}$$

Thus, 'c' can be marked.

- V_{dc} will be \perp to CD. Through 'c', a line \perp to CD is drawn. V_{do} will be \parallel to the path of motion of D. Through 'o', a line \parallel to the path of motion of D is drawn. The intersection of these lines locates 'd'.

$$\text{From the vel. diagram, } \omega_{qc} = \frac{V_{cq}}{r(QC)} = \frac{1.4}{0.15}$$

$$\therefore \omega_{qc} = 9.33 \text{ rad/s } \curvearrowright$$

Acceleration diagram

Sr. No.	Vector	Magnitude	Direction	Sense
1	a_{ao}^c	$= \frac{v_{ao}^2}{(AO)} = \frac{(2.51)^2}{(0.2)} = 31.5 \text{ m/s}^2$	// to OA	Towards O in confg. diag.
2	a_{ao}^t	$= 0 \quad \therefore \alpha_{ao} = 0$	—	—
3	a_{aia}^{cor}	$= (2 v_{aia} \omega_{qc}) = 2(0.85)(9.33) = 15.86 \text{ m/s}^2$	As shown in fig (\perp to BC)	—
4	a_{aia}^s	$= 6.9 \text{ m/s}^2$	// to BC	—
5	a_{dc}^c	$= \frac{v_{dc}^2}{r(CD)} = \frac{(0.95)^2}{(0.5)} = 1.81 \text{ m/s}^2$	// to CD	Towards C in fig.
6	a_{dc}^t	$= r(CD) \alpha_{dc} = 11.75 \text{ m/s}^2$ $\therefore \alpha_{dc} = \frac{11.75}{0.5} = 23.5 \text{ rad/s}^2$	\perp to a_{dc}^c	—
7	a_{do}	$= 6.75 \text{ m/s}^2$	// to path of motion of D	—
8	a_{aig}^c	$= \frac{v_{aig}^2}{r(AIQ)} = \frac{(2.45)^2}{(0.2625)} = 22.87 \text{ m/s}^2$	// to AIQ	Towards Q in fig.
9	a_{aig}^t	$= r(AIQ) \alpha_{aig} = 5.65 \text{ m/s}^2$ $\therefore \alpha_{aig} = \frac{5.65}{(0.2625)} = 21.52 \text{ rad/s}^2$	\perp to a_{aig}^c	—

1) a_{ao}^c is drawn. a_{ao}^t is zero. Thus, 'a' is located.

2) a_{aia}^{cor} is drawn. a_{aia}^s will be \perp to a_{aia}^{cor} . Through 'a', a line \perp to a_{aia}^{cor} is drawn. (line 1).

3) a_{aig}^c is drawn. a_{aig}^t will be \perp to a_{aig}^c . Through 'aig', a line \perp to a_{aig}^c is drawn. (line 2)

The intersection of these two lines locates point 'ai'.

4) 'aig' is extended to point 'c', such that

$$\frac{qc}{aig} = \frac{QC}{AIQ} \quad \therefore qc = \frac{(4.75)}{\text{cm}} \left(\frac{150}{262.5} \right)$$

$$\therefore qc = 2.71 \text{ cm}$$

Thus, 'c' can be marked.

5) a_{dc}^c is drawn. a_{dc}^t will be \perp to a_{dc}^c . Through 'dc', a line \perp to a_{dc}^c is drawn. (line 1)

6) a_{do} will be \parallel to the path of motion of D.
Through 'o', a line \parallel to the path of motion of D is drawn. (line 2)

The intersection of these two lines locates point 'd'.

From the accⁿ diagram

1) Accⁿ of block D = $a_{do} = 6.75 \text{ m/s}^2$

2) Angular accⁿ of slotted bar QB
= α_{qb}
= $\alpha_{a1q} = 21.52 \text{ rad/s}^2$ ↷

(P13) In the mechanism shown in the figure, the crank AB, 70 mm long, rotates clockwise at 110 rpm. CD is 140 mm long. Link BD is 260 mm long, and slides through a swivelling pin E at the lower end of rod EF. EF slides in vertical guides as shown.

For the position shown, find the angular velocity and angular acceleration of DB, and linear velocity and linear acceleration of F.

Solution - Velocity diagram -

$$1) V_{ba} = \omega(AB) = \frac{2\pi(110)}{60} = (0.07) \frac{2\pi(110)}{60} = 0.81 \text{ m/s.}$$

2) Draw V_{ba} .

3) V_{db} will be \perp to BD. Through 'b', draw a line \perp to DB. V_{dc} will be \perp to CD. Through 'c', draw a line \perp to CD. The intersection locates 'd'.

4) On 'bd', mark 'e', such that $\frac{de_1}{db} = \frac{DE_1}{DB}$

$$\therefore de_1 = \left(\frac{3.75}{\text{cm}}\right) \left(\frac{76}{260}\right) = 1.1 \text{ cm.}$$

Thus 'e' can be marked.

{ E_1 is a point coincident with E, but on link BD}

5) V_{ee_1} will be \parallel to DB. Through 'e', draw a line \parallel to BD. V_{eo} will be \parallel to the path of F. Through 'a', draw a line \parallel to the path of F. The intersection locates point 'e'.

From the velocity diagram,

$$1) \text{ Angular velocity of link DB} \\ = \omega_{db} = \frac{V_{db}}{\omega(AB)} = \frac{0.375}{(0.26)} = 1.44 \text{ rad/s} \curvearrowright$$

$$2) \text{ Linear velocity of F} \\ = \underline{V_f = V_{ea} = 0.45 \text{ m/s}}$$

Acceleration diagram

Sr. No.	Vector	Magnitude	Direction	Sense
1	a_{ba}^c	$= \frac{V_{ba}^2}{r(BA)} = \frac{(0.81)^2}{(0.07)} = 9.37 \text{ m/s}^2$	// to AB	Towards A in config. diag.
2	a_{ba}^t	$= 0 \quad \therefore \alpha_{ab} = 0$	—	—
3	a_{db}^c	$= \frac{V_{db}^2}{r(DB)} = \frac{(0.375)^2}{(0.26)} = 0.54 \text{ m/s}^2$	// to DB	Towards B in config. diag.
4	a_{db}^t	$= r(DB)\alpha_{db} = 8.1 \text{ m/s}^2$ $\therefore \alpha_{db} = \frac{8.1}{0.26} = 31.15 \text{ rad/s}^2$	\perp to a_{db}^c	—
5	$a_{ee_1}^{\text{cor.}}$	$= 2(V_{ee_1})\omega_{db} = 2(0.53)(1.44) = 1.53 \text{ m/s}^2$	As shown in fig. (\perp to BD)	—
6	$a_{ee_1}^s$	$= 8.3 \text{ m/s}^2$	\perp to $a_{ee_1}^{\text{cor.}}$	—
7	a_{ea}	$= 2.55 \text{ m/s}^2$	// to path of F.	—
8	a_{dc}^c	$= \frac{V_{dc}^2}{r(DC)} = \frac{(0.53)^2}{(0.14)} = 2.01 \text{ m/s}^2$	// to CD	Towards C in config. diag.
9	a_{dc}^t	$= r(DC)\alpha_{dc} = 8 \text{ m/s}^2$ $\therefore \alpha_{dc} = \frac{8}{0.14} = 57.14 \text{ rad/s}^2$	\perp to a_{dc}^c	—

1) Draw a_{ba}^c .

2) Draw a_{db}^c . a_{db}^t will be \perp to a_{db}^c . Through 'd_b', a line \perp to a_{db}^c is drawn. (line 1)

3) Draw a_{dc}^c . The a_{dc}^t will be \perp to CD. Through 'd_c', a line \perp to a_{dc}^c is drawn. (line 2)

The intersection of these lines locates 'd'.

4) On 'd_b', a point 'e₁' is marked such that

$$\frac{de_1}{db} = \frac{DE_1}{DB} \quad \therefore de_1 = \left(\frac{76}{260}\right)_{\text{cm}} (8.1) = 2.37 \text{ cm.}$$

Hence 'e₁' is marked.

5) Draw $a_{ee_1}^{\text{cor.}}$. $a_{ee_1}^s$ will be \perp to $a_{ee_1}^{\text{cor.}}$. Through 'e₁', a line \perp to $a_{ee_1}^{\text{cor.}}$ is drawn. (line 1)

6) a_{ea} will be // to path of F. Through 'a', a line // to path of F is drawn. (line 2)

1a) The intersection of these lines locates point

From the accⁿ diagram,

1) Ang. accⁿ of link DB
 $= \alpha_{db} = 31.15 \text{ rad/s}^2 \curvearrowright$

2) linear accⁿ of F
 $= \frac{a_{fa}}{a_{ea}} = 2.55 \text{ m/s}^2$
